

Evaluation of DyMESH Wheel Impact in a Rollover Collision

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Goal

To evaluate the performance of the wheel impact model within SIMON DyMESH by simulating a rollover crash test involving significant interaction between the wheel of the bullet vehicle and the body of the target vehicle.



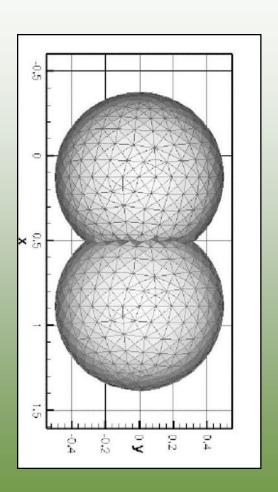
Outline

- Review of SIMON/DyMESH Wheel Impact Model
- Full Scale Test Configuration and Simulation Set-Up
- 3. Simulation Results





- DyMESH: "Dynamic MEchanical Shell".
- Wheel impact model introduced in SIMON Version 3 within HVE Version 9, in April 2012
- Allows DyMESH simulations involving impacts between the wheels and sprung masses of <u>other</u> vehicles
- Still considered "beta" software pending further user feedback



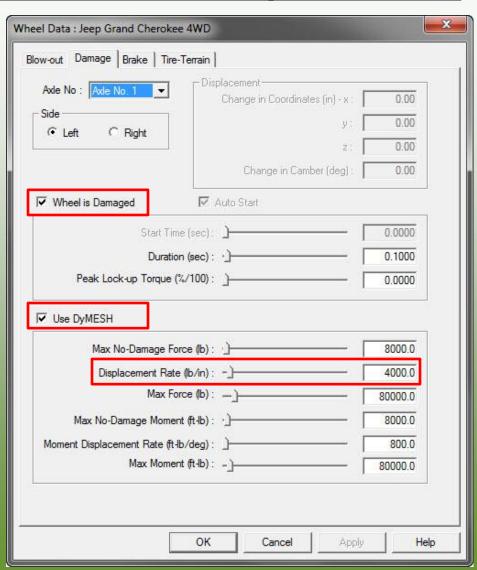


- Each wheel assembly is a DyMESH object with mass and a mesh
- Properties of the wheel assembly are defined by tire dimensions, inertia and radial stiffness
- Tire kinematics are included: spinning wheels create moments and forces from tangential friction
- The resulting forces and moments are incorporated into the equations of motion
- Forces and moments on wheels can cause displacement, based on suspension stiffness properties



Enabling DyMESH modeling of wheel impact:







EDC DyMESH wheel impact model demonstration:

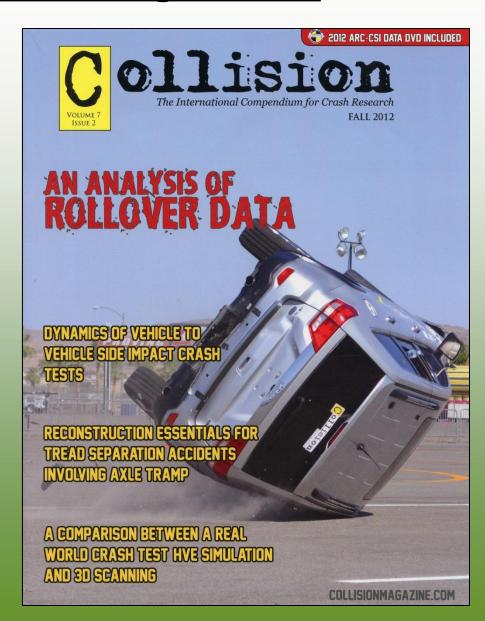




2. Full Scale Test Configuration and Simulation Set-Up



- Test conducted by W.R. Haight at 2012 ARC-CSI Crash Conference
- 2011 Jeep Grand
 Cherokee induced to roll
 over by driving up and
 over 1987 Bertone ("Fiat")
 X1/9 sports car







- 2011 Jeep Grand Cherokee
- 5,080 lb test weight

- Radio-controlled
- 3-axis accelerometers and rate gyros

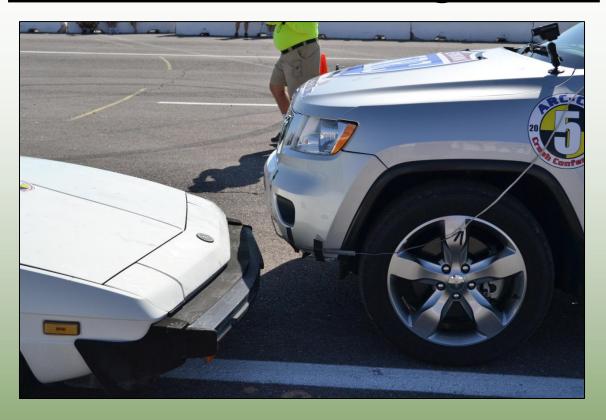




- 1987 Bertone X1/9
- 2,010 lb test weight

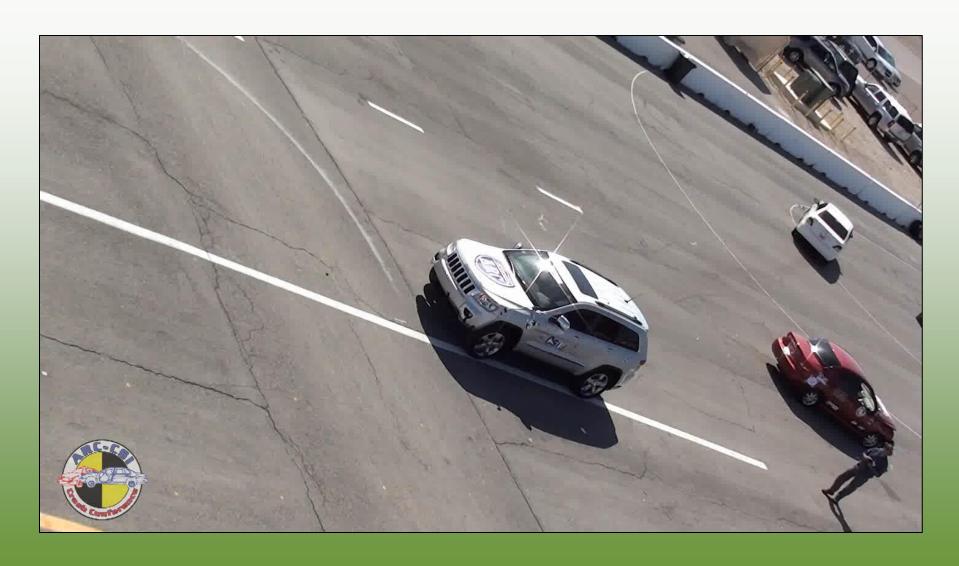
- Parked in neutral
- 3-axis accelerometer



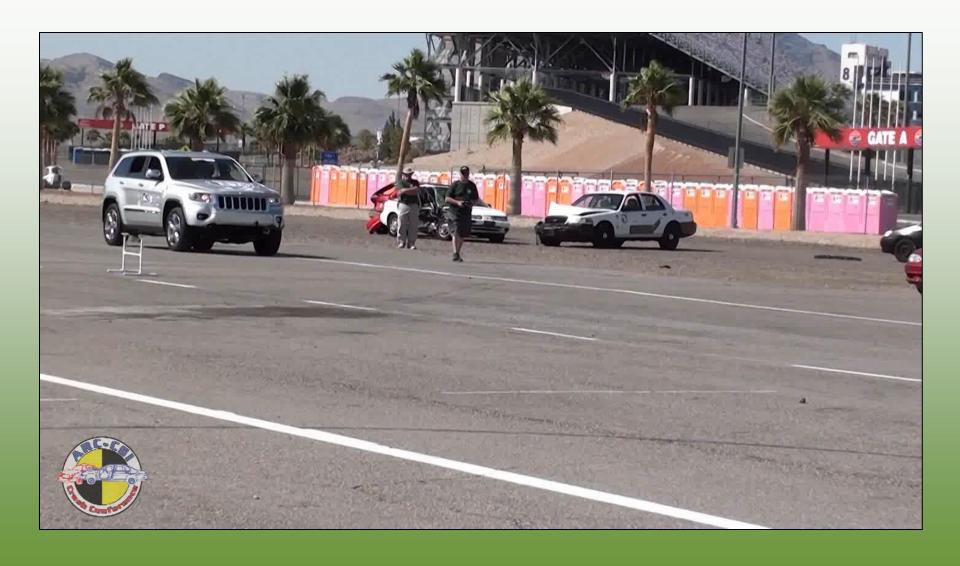


- Offset frontal impact (half width of Fiat)
- ~41 mph impact speed for Jeep

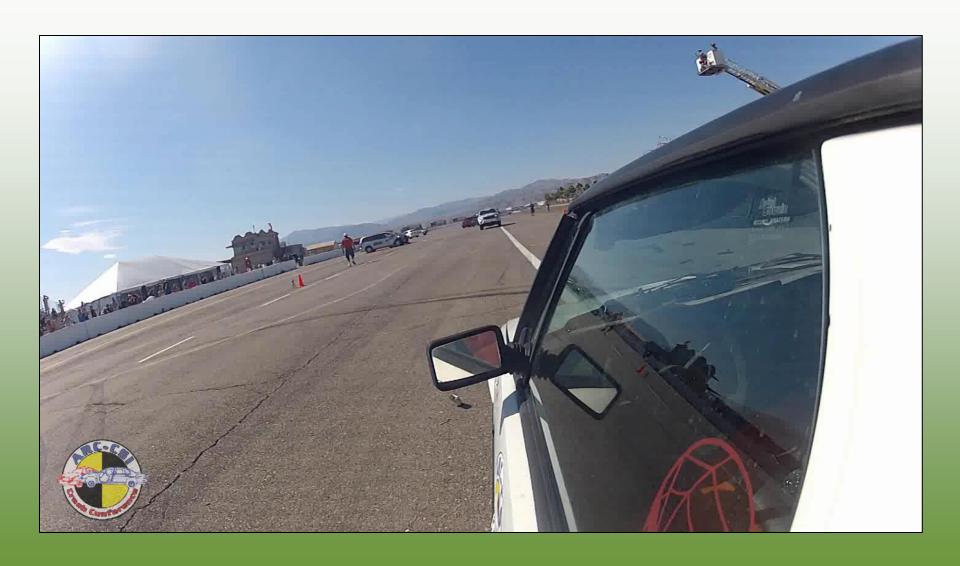














Simulation Set-Up



- 2011-2012 Jeep Grand Cherokee from Vehiclemetrics database
- DyMESH wheel displacement rate increased from default



Simulation Set-Up



- Vehicle data based on Class 1 Generic Vehicle
- Body geometry from 3D CAD Browser, matched against surveyed test vehicle
- Front and top body stiffness increased via reduction of conversion height from 30" to 8" (yes, there is a frontal crash test for this vehicle!)

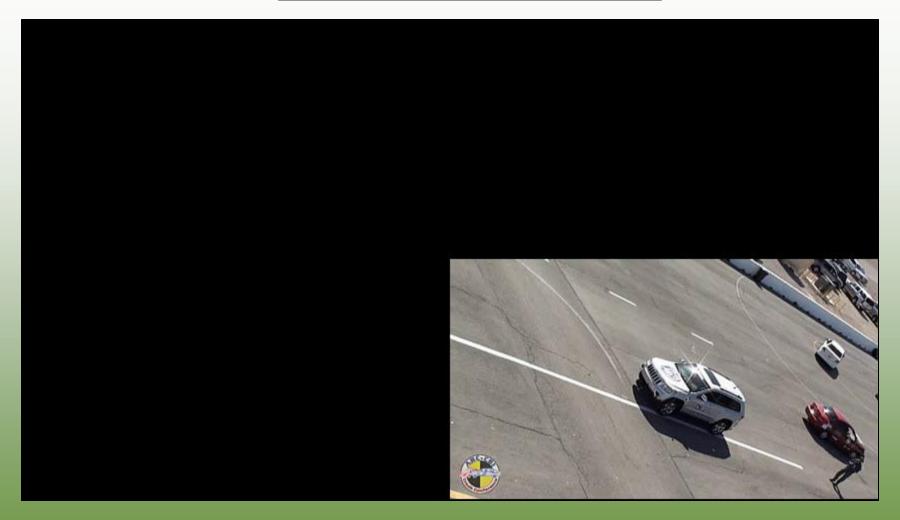


Simulation Set-Up

- Integration timesteps = 0.001 sec
- Point contact tire model active, only
- Results in 1 ¼-minute run time for 2 ½-sec simulation on 3.40 GHz PC with 12 GB RAM running Windows 7 64-bit + Nvidia GeForce GT420 video accelerator (2 GB RAM)









t = 0.01 sec



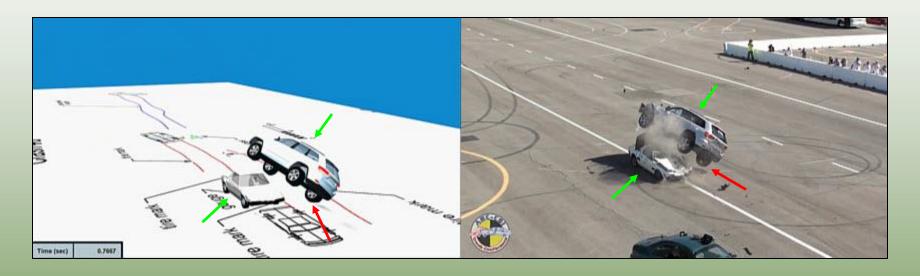


t = 0.11 sec



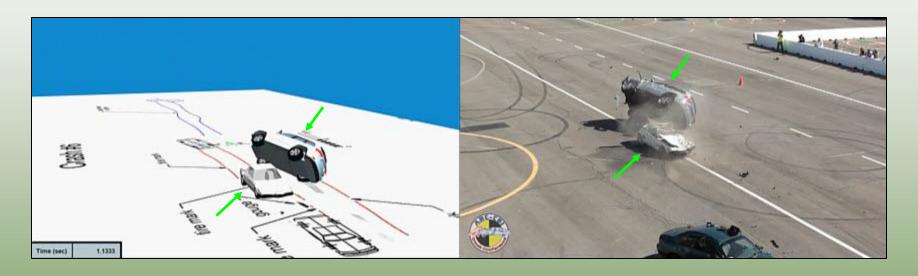


t = 0.51 sec



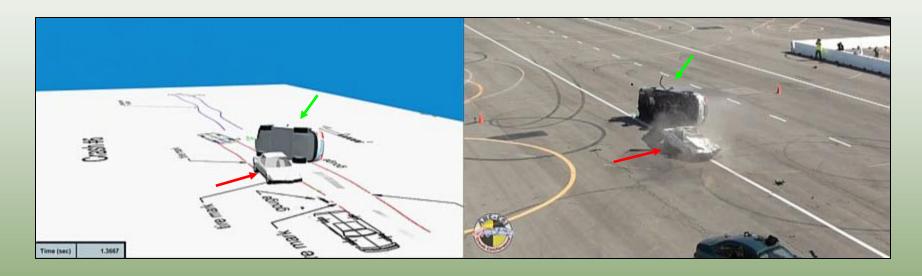


t = 0.88 sec

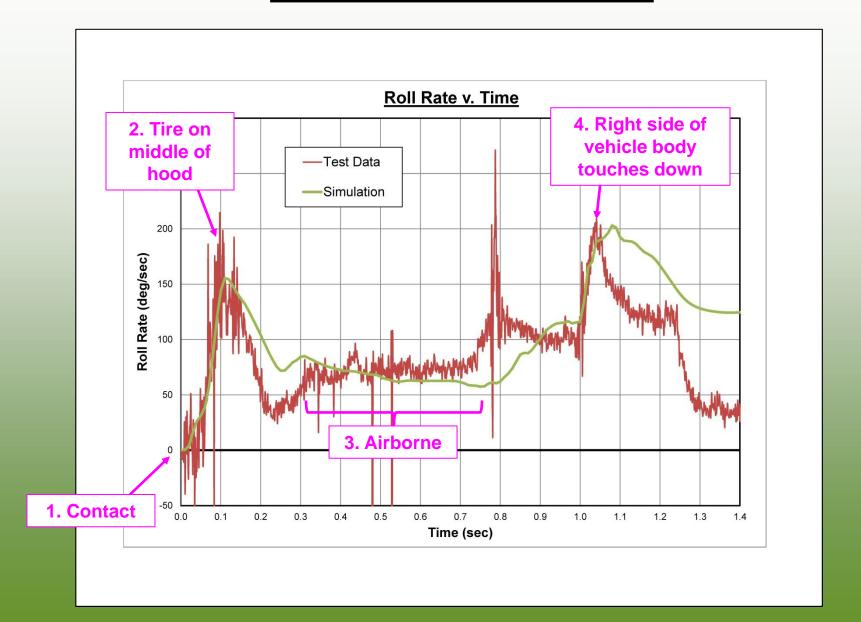




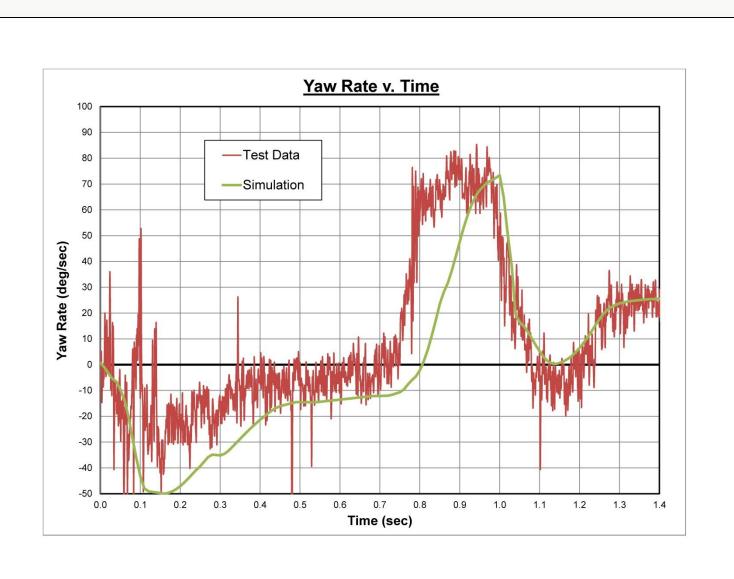
t = 1.10 sec



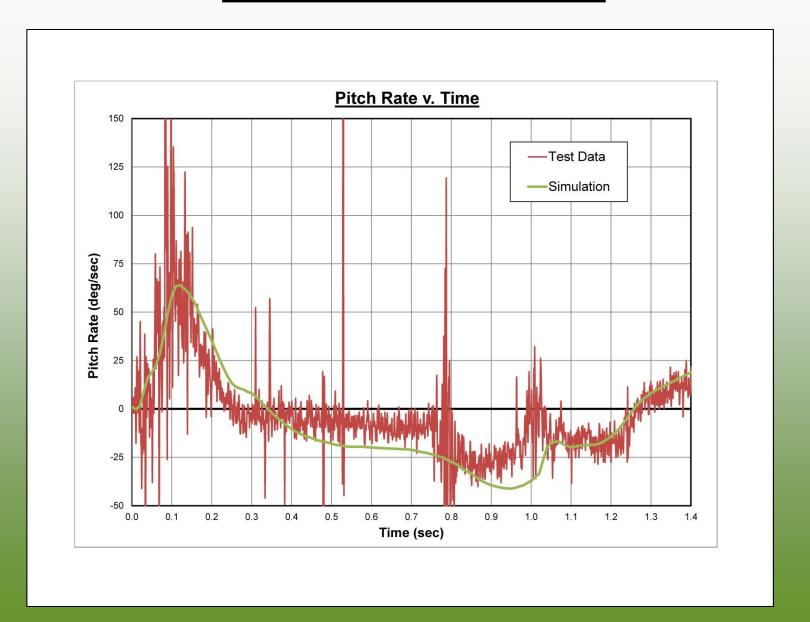




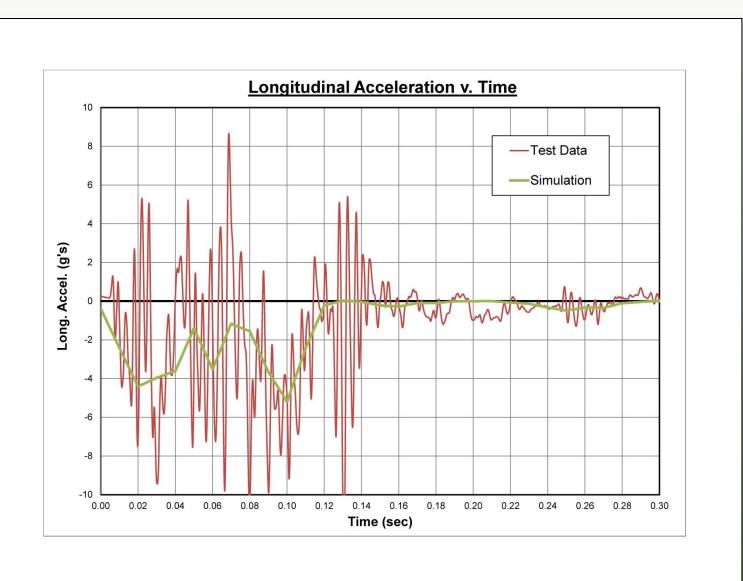




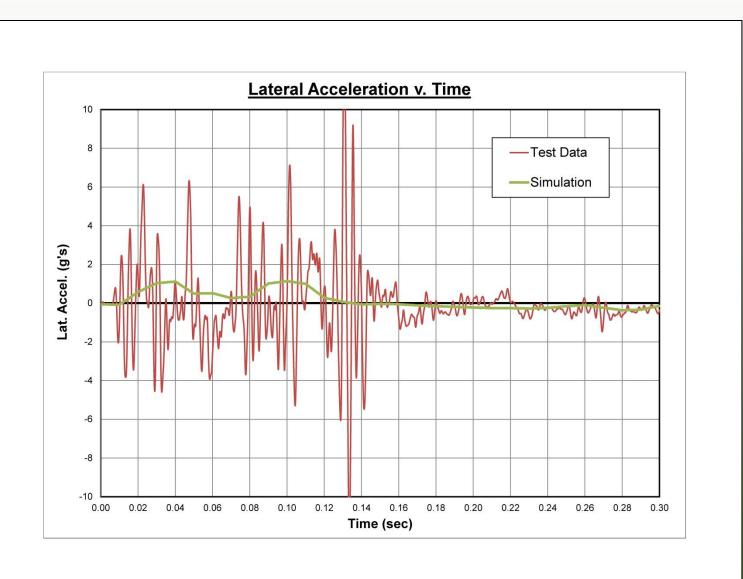




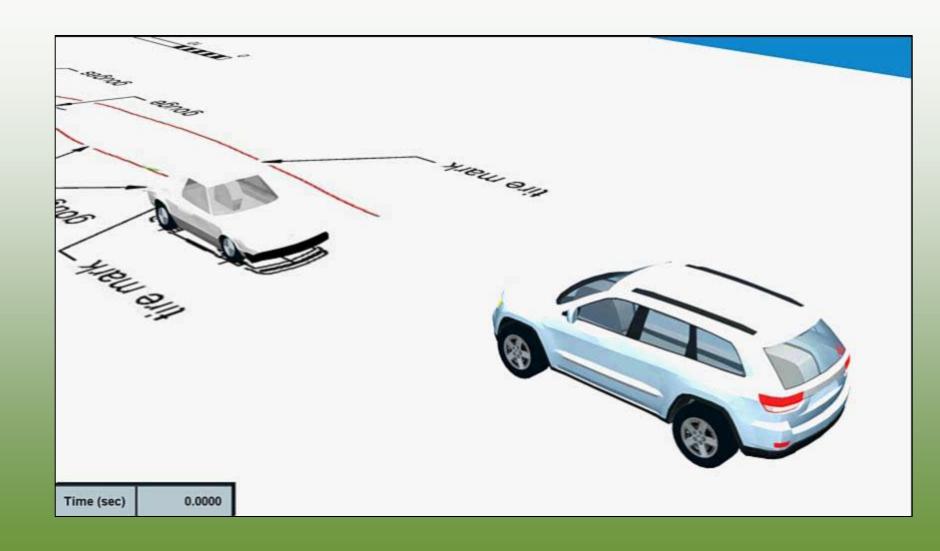














<u>Summary</u>

- In an initial comparison study, the DyMESH wheel impact model functioned to properly alter the path of the bullet vehicle as it overran the body of the target vehicle
- The primary parameters examined in this study were the stiffness of the body of the target vehicle and the force-deflection relationship of the wheel assembly of the bullet vehicle
- The proper simulation of the entirety of the rollover event was not a focus of this study, however, the bullet vehicle's behavior was adequately modeled from impact through separation from the target vehicle
- Modeling detail for the interaction between a wheel and the body of its own vehicle may affect the trajectory of an overrunning bullet vehicle.
- It is expected that further refinement of this initial study would improve the model correlation (e.g. linear accelerations and angular rates.)



Acknowledgement

 The authors greatly acknowledge W.R. "Rusty" Haight of Collision Safety Institute for undertaking the subject rollover test and for providing the authors with the highquality documentation and test data upon which this simulation study was based



Questions?

